



Comprehensive Comparisons of Two- and Three-Dimensional Numerical Estimation of Stress Intensity Factors and Crack Propagation in Linear Elastic Analysis

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DOI: <https://doi.org/10.30880/ijie.2019.11.06.006>

Received 12 May 2019; Accepted 31 August 2019; Available online 06 September 2019

Abstract: This paper presents a comprehensive comparison of two finite element software in two and three dimensions. The first is a two-dimensional crack growth simulation program developed as source code by using Visual Fortran Language whereas the second is ANSYS mechanical program (3D structural analysis for fracture mechanic analysis). The procedure consists of computing stress intensity factors (SIFs), the crack growth path and the stresses and strain distributions via an incremental analysis of the crack extension, considering two and three-dimensional analysis. Comparisons between this two software were performed for different case studies for stress intensity factors as well as crack growth trajectory. This source code includes the mesh generator based on the advancing front method as well as all the pre and post process for the crack growth simulation under linear elastic fracture mechanics theory with a user-friendly interface. The maximum circumferential stress criterion was used for prediction of the crack growth in isotropic materials under mixed-mode loading. Furthermore, the equivalent domain integral method has been used for calculating the stress intensity factors values during crack growth. Verification of the predicted crack path and stress intensity factors values were validated with relevant experimental data and numerical results obtained by other researchers with a good agreement.

Keywords: Finite element method, advancing front method, ANSYS Mechanical, Stress Intensity Factors, Linear elastic fracture mechanics

1. Introduction

Reliable structural integrity and reliability assessment of the critical structural components remain an essential process for the safety of engineering structures including crack initiation and propagation. Stress intensity factors are important parameters for failure analysis of cracked structures because it has been used for the prediction of crack initiation and cracks propagation path. Stress intensity factors (SIFs) defines the amplitude of the crack tip singularity. These stresses near the crack tip increase in proportion to SIFs. Moreover, the stress intensity factor completely defines the crack tip conditions; if SIFs are known, it is possible to solve for all components of stress, strain, and displacement. Numerous handbooks of stress intensity factors calculation are available [1,2] for specific geometries and loading. Due to the limitation of the analytical solution of stress intensity factors, the vast majority of fracture problems encountered in engineering practice should be resolved with a numerical analysis method [2]. Stress intensity factors can be computed by various methods, such as the boundary element method (BEM), finite element method (FEM) by[4], both FEM and BEM by[5]. The basic practical problem facing a designer is to make a decision as to the method for determining stress intensities.

There exist two categories of commercial FEM programs, finite element programs such as ANSYS [6,7], ABAQUS, NASTRAN, etc. can be used to add elements manually and perform analysis on complex structures. The other type is professional FEM programs, such as NASGROW, AF-GROSS, etc. these are more expensive but provide a high precision calculation [8]. However, there are limitations when it comes to more complicated geometries and

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loading conditions. This is due to the density of the generated mesh as well as the element type used. Other researchers developed their own two-dimensional source code program to predict the fatigue crack growth and crack propagation under static loading as well as the prediction of stress intensity factors using adaptive mesh strategy [9,10,11]. It is not easy to strike a balance between the accuracy of the method, the time required to get a solution, and cost. Numerous equations for stress intensity factors are available in the literature [12, 13]. These factors represent various geometries and loading conditions of fundamental importance in the prediction of structural failure of cracked bodies. In all, there are probably more than 600 formulas for calculating K values for different crack configurations, body geometries, and loading situations. However, it appears that the bulk of fracture mechanics work to date has been limited to a single-mode loading largely because little is known about mixed-mode phenomena [14]. For these reasons, the evaluation of stress intensity factors for non-standard geometries are required by using numerical methods like finite element methods.

Many studies have been performed to predict the stress intensity factors as well as the crack growth path like [15,16,17]. This study presents a comprehensive comparison study for crack growth between the developed source code two dimensional and three-dimensional FEM (ANSYS Mechanical). The fracture parameters such as stress intensity factors and crack growth trajectory were compared on different types of geometries under specified loading condition. The predicted results were validated by the corresponding experimental values for the crack growth trajectory.

2. Developed Program Criteria and Mesh Refinements

In this work, the unstructured triangle mesh is automatically generated by using the advancing front method [18]. For more accurate control for the geometrical characteristics such as the element size, element stretching and stretching directions for the new mesh, the background mesh should be generated. Here the strategy taken to generate the background mesh is to utilize all the initial boundary nodes of geometry and construct the boundary triangles as the background mesh by the dichotomy technique [19, 9]. Adaptive subdomain remeshing techniques add each crack advance segment by modifying the current finite element mesh. This is done by removing elements and nodes in a region ahead of the current crack tip and replacing them with smaller elements along the crack faces to be added. These additional elements typically include a small structured rosette of elements at the new crack tip location. The remaining space between is filled with additional elements Insufficiently robust mesh modification algorithms can cause premature termination of a progressive fracture simulation.[20]. Merely a geometric updating of a crack path during each simulation step because cracks are not represented explicitly in an adaptive mesh. This direct method has been extended to represent multiple cracks without the need for mesh refinement between crack tips [21]. Namely, there are three criteria used for prediction of the crack path, the maximum circumferential stress criterion, the maximum energy release rate criterion and the minimum strain energy density criterion [22]. In the present study, the maximum circumferential stress criterion is exploited for isotropic materials under mixed-mode loading [9,11]. In the maximum circumferential stress criterion, crack propagation is observed in the normal direction of maximum tangential tensile stress given by:

$$\sigma_{\theta} = \frac{1}{\sqrt{2\pi r}} \cos \frac{\theta}{2} \left[K_I \cos^2 \frac{\theta}{2} - \frac{3}{2} K_{II} \sin \theta \right] \quad (1)$$

The normal direction to the maximum tangential stress can be obtained by solving $d\sigma_{\theta}/d\theta=0$ for θ . The nontrivial solution is expressed as:

$$K_I \sin \theta + K_{II} (3\cos \theta - 1) = 0 \quad (2)$$

Where the crack growth angle θ is given by:

$$\theta = \pm \cos^{-1} \left\{ \frac{3K_{II}^2 + K_I \sqrt{K_I^2 + 8K_{II}^2}}{K_I^2 + 9K_{II}^2} \right\} \quad (3)$$

In 1968 Rice introduced J-integral to study non-linear material behavior in small scale yielding. It is a path independent contour integral defined as:

$$J = \int_c \left[W n_1 - \sigma_{ij} n_j \frac{\partial u_i}{\partial x} \right] ds \quad (4)$$

where strain-energy density is denoted by W ; stresses by σ_{ij} ; the displacements corresponding to local i -axis u_i ; arc length of the contour is expressed as s ; n_j is the unit outward normal to the contour C , surrounding the crack tip (Figure 1a).



Figure 1: (a) Arbitrary contour containing the crack tip; (b) Area to calculate the J-integral

The Equivalent Domain Integral Method being more convenient for finite element analyses replaces finite size domain by the divergence theorem by the integration along the contour. For two-dimensional problems area integral replaces the contour integral i (Figure 2b). Then, equation (4) is rewritten as:

$$J_k = -\int_A \left[W \frac{\partial q}{\partial x} - \sigma_{ij} \frac{\partial u_i}{\partial x} \frac{\partial q}{\partial x} \right] dA - \int_A \left[\frac{\partial W}{\partial x} - \frac{\partial}{\partial x} \left(\sigma_{ij} \frac{\partial u_i}{\partial x} \right) \right] q dA - \int_s t_i \frac{\partial u_i}{\partial x} q ds \quad (5)$$

In linear elastic analysis, the J-integral by definition take account of translational mechanical energy balance in front of the crack along the x -axis. In either cases of pure Mode I and Mode II, equation (5) facilitate the calculation of the stress intensity factors K_I or K_{II} but fails for mixed mode condition to calculate K_I and K_{II} separately. For such case invariant integrals are used as defined by Knowles and Sternberg [23] as:

$$J_k = -\int_A \left[W \frac{\partial q}{\partial x_k} - \sigma_{ij} \frac{\partial u_i}{\partial x_k} \frac{\partial q}{\partial x_j} \right] dA - \int_A \left[W \frac{\partial q}{\partial x_k} - \sigma_{ij} \frac{\partial}{\partial x_j} \left(\frac{\partial u_i}{\partial x_k} \right) \right] q dA - \int_s t_i \frac{\partial u_i}{\partial x_k} q ds \quad (6)$$

where k is an index for local crack tip axis (x, y). These integrals were introduced initially for small deformation and were extended by Atluri [24] for finite deformation. The stress intensity factors can be obtained by two possible ways. The first approach is through relationships between the J-integral and the stress intensity factors. These relations are:

$$\begin{aligned} J_1 &= \frac{\kappa+1}{8\mu} (K_I^2 + K_{II}^2) \\ J_2 &= -\frac{\kappa+1}{4\mu} K_I K_{II} \end{aligned} \quad (7)$$

The second approach is through the relations between the stress intensity factors and J_1, J_2 as:

$$\begin{aligned} K_I &= 0.5 \sqrt{\frac{8\mu}{\kappa+1}} (\sqrt{J_1 - J_2} + \sqrt{J_1 + J_2}) \\ K_{II} &= 0.5 \sqrt{\frac{8\mu}{\kappa+1}} (\sqrt{J_1 - J_2} - \sqrt{J_1 + J_2}) \end{aligned} \quad (8)$$

3. Simulation Results and Discussion

In this section, the crack path and other fracture parameters will be computed for seven modified compact tension specimens with different geometrical parameters based on the hole position as shown in table 1. The simulation for each geometry is performed in both developed source code and ANSYS Mechanical. In this study, the tested material is 7075-T6 aluminum alloy, which is widely used in the manufacturing of structural components in the aerospace industry. The material Young's modulus is 72 GPa, and its Poisson's ratio is 0.3, as well as a material fracture toughness K_C of 29.66 MPa m^{1/2}. The graphical representation of the specimen is shown in Figure 2, and the thickness is 4 mm. For the hole with d_I diameter, the horizontal distance from its center to the center of semi-circle is equal to h_I ,

and the shortest distance from its edge to the symmetrical axis of specimen equals v_1 . For the hole with d_2 diameter, its corresponding values are h_2 and v_2 , respectively [24].

Table 1: Geometrical parameters for specimens.

Specimen	Hole parameters (Zhang, Zhang et al. 2016)[24]					
	d_1	h_1	v_1	d_2	h_2	v_2
G1
G2	8	6.5	3
G3	8	10.5	3
G4	8	14.5	3
G5	2	6.5	3
G6	4	6.5	3
G7	5	6.5	3	5	6.5	3

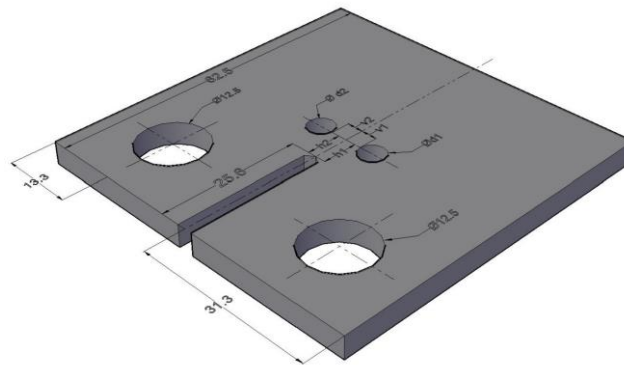


Figure 2: Schematic of the specimen

To evaluate the effect of holes on the fatigue crack growth path, a series of specimens with different holes are obtained with altering its sizes and locations, which are shown in Table 1 in details. In the first set in Table 1, two specimens are utilized. One specimen without any hole is referred to as G1, and the other with a drilling hole is denoted as G2. For specimen G2, the hole diameter d_1 is 8 mm. Its horizontal distance h_1 and vertical distance v_1 are 6.5 mm and 3 mm, respectively. In the top 12.3 mm hole of each geometry a constant amplitude loading of 3 KN with cyclic load ratio $R=0$ is applied, whereas the structure is clamped at the bottom 12.3 mm hole and the displacements and rotations of all nodes along its boundary were fully constrained. The fatigue crack growth path predicted by Ansys Workbench and the developed source code program for G1 is depicted in Figure (3) and compared with the experimental and numerical results using Franc3D software performed by [24] which almost identical. The number of cycles computed by the Ansys Program for G1 was 44803 with a crack length of 18.14 mm while in Franc3D ((Zhang, Zhang et al. 2016 [24])) it was 11765 cycles and a crack path length of 11.9 mm.

The quality of the mesh plays a significant role in the accuracy and stability of the numerical computation. Regardless of the type of mesh used in your domain, checking the quality of your mesh is essential. One important indicator of mesh quality that ANSYS allows you to check is a quantity referred to as the orthogonal quality. The mesh quality is always dependent on the aspect ratio. The aspect ratio is a measure of the stretching of a cell. It is computed as the ratio of the maximum value to the minimum value of any of the following distances: the normal distances between the cell centroid and face centroids (computed as a dot product of the distance vector and the face normal), and the distances between the cell centroid and nodes. The aspect ratio is differed based on the mesh density which will be higher in the area around the crack tip for accurate prediction of stress intensity factors and other stresses distribution. The mesh generated for the G1 in Present developed program is 4968 nodes with 2341 elements while for Ansys it is 160678 nodes and 112090 elements. Figure 4 shows the calculated values of stress intensity factors for mode I and mode II (K_I and K_{II}) with respect to the crack length. There is approximately 3% difference between the mode values computed for both programs. The range of K_I in Present developed program is 624 to 1068 MPa.mm^{0.5} while in Ansys it ranges between 587 to 998 MPa.mm^{0.5}.

In the G2 geometry a hole 8 mm in diameter is introduced. It is placed at $h_1= 6.5$ mm away from the crack tip, and from the center line $v_1= 3$ mm, figure 4.30. The K_I of all nodal points along the crack front goes beyond the K_C mentioned, and the steady fatigue crack growth length reaches 8.2 mm in Ansys and 7.29 mm in the present developed program, and 4.5 mm in Franc3D as shown in Figure 5. The Von Mises stress computed for Ansys differs from that of the work of [24] due to length of the crack. In Ansys, the crack reaches the maximum it could, that is why we see the distribution of stress is high at the crack tip and at the circumference of the 8 mm hole. The crack produced in the G2 specimen changes as it continues to grow. Furthermore, this shows that when a small hole is near the path at which the crack

moves through, the crack will propagate towards to the hole and so the fatigue crack path deviates due to the attraction of the hole as shown in Figure 5.

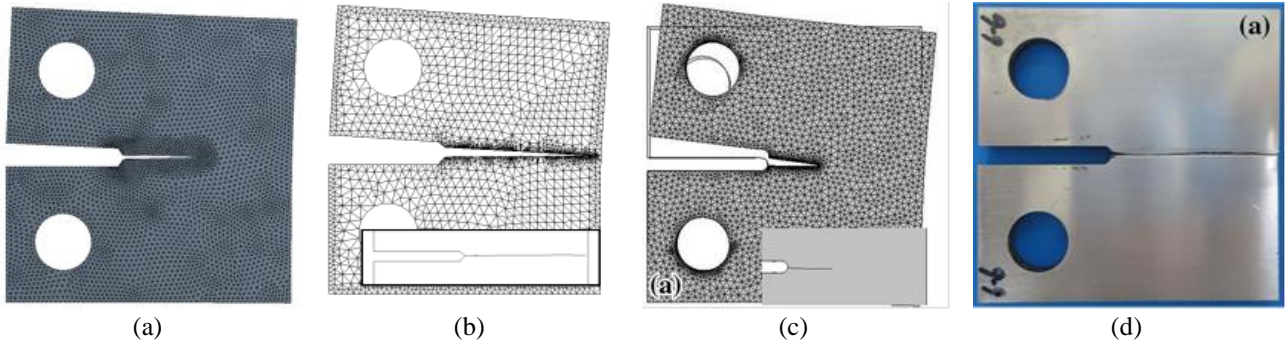


Figure 3: G1 deformed shape and crack path: a) Ansys WB., b) Present developed program, c) Franc3D [24], d) Experimental Work [24]

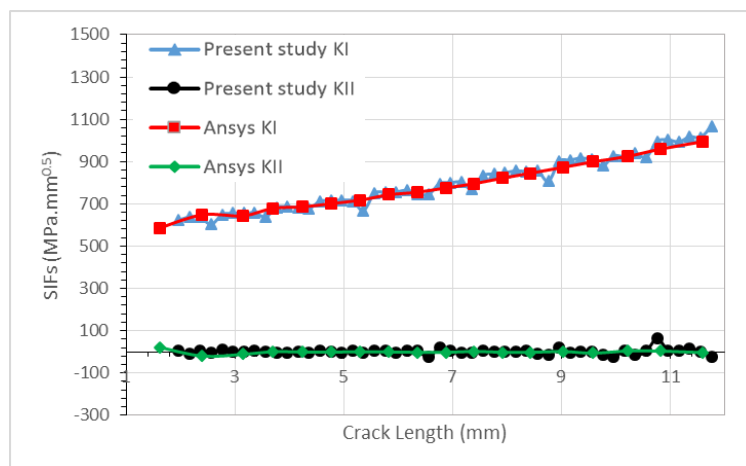


Figure 4: SIFs relation with the crack length, specimen G1.

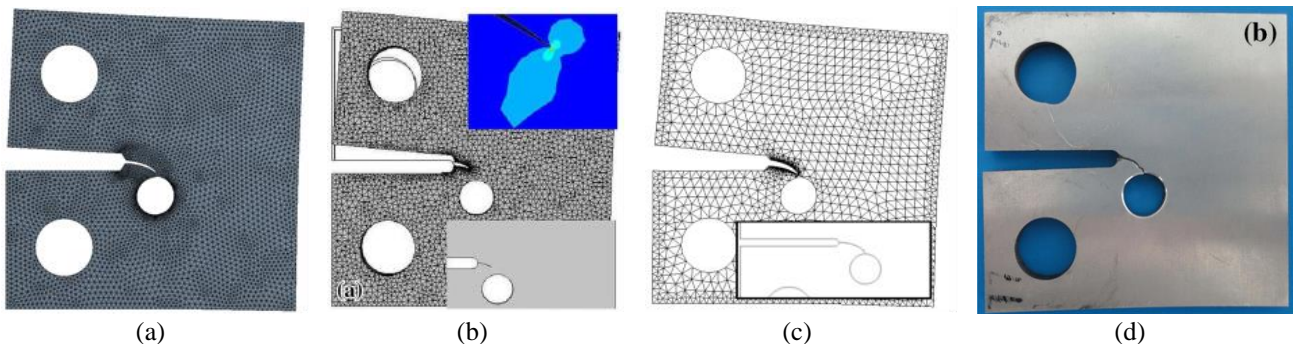


Figure 5: G2 deformed shape, equivalent von mises, crack path: a) Ansys, b) Present developed program, c) Franc3D [24], d) Experimental work [24].

For the simulation of specimen G2, there is a 0.57% difference in the stress intensity factors computed between the Ansys program and the developed program as shown in Figure 6. The mesh density in the present developed program is 4955 nodes and 2327 elements while in Ansys, 176831 nodes and 120910 elements. To further evaluate the effect of a small holes on the crack growth direction, G3 specimen has an 8 mm diameter hole at position $h_1 = 10.5$ and v_1 is the same as G2 (Figure 5). The crack path predicted are identical in all three programs as well as the experimental work performed by Zhang et al. (Figure 7). Yet again since the hole gets closer to the crack path direction, the crack starts to propagate towards the hole. The fatigue crack growth finished the simulation in Ansys with a total number of cycles of 70751 and a crack length of 11.482 mm. In Present developed program the simulation completed with a crack length of 6.789 mm while in Franc3D the simulation completed with 3818 cycles and a crack length of 5.8 mm. With a difference of 0.81%, it can be seen from Figure 8 that the stress intensity factor modes one and two is almost identical in both applications for specimen G3. The simulation performed for G3 was done with 4947 nodes and 2323 elements in Present developed program, in Ansys it was done with a mesh density of 155911 nodes and 106740 elements.

Figure 9 shows the effect of the crack path when the hole becomes further away from the crack front. In specimen G4, the 8 mm diameter is positioned at a distance, $h_1 = 14.5$ mm. It can be noticed for the G4 specimen that the crack path started straight and since the hole is in the path of the crack, the path deviated towards the hole. The predicted paths in both Ansys and Present developed program is almost a replica of the experimental and Franc3D work done by respectively [24]. The Ansys predicted values are 46314 cycles and a crack length of 14.618 mm while Franc3D values were 4716 cycles and a 6.7 mm crack length obtained by [24]. The K_I range in Present developed program for specimen G4 is from 607 to 1072 $\text{MPa}\cdot\text{mm}^{0.5}$, for Ansys it ranges between 478 to 1196 $\text{MPa}\cdot\text{mm}^{0.5}$ (Figure 10). This difference is not an error difference but rather the K_I at the initial crack length and the crack tip at the end of the crack length. There is a 1.19 % difference between the mode 1 values at the same crack length. The mesh generated for the G4 specimen in Present developed program is 4943 nodes and 2321 elements while in Ansys a mesh with element size 2 mm, 166881 nodes, and 114429 elements.

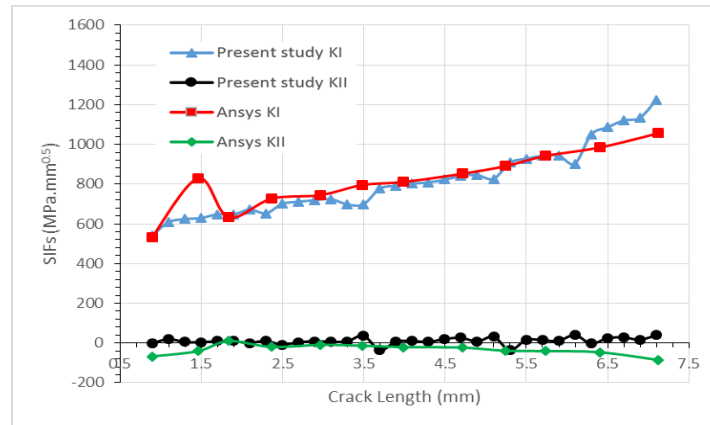


Figure 6: G2, stress intensity factors relationship with the crack length.

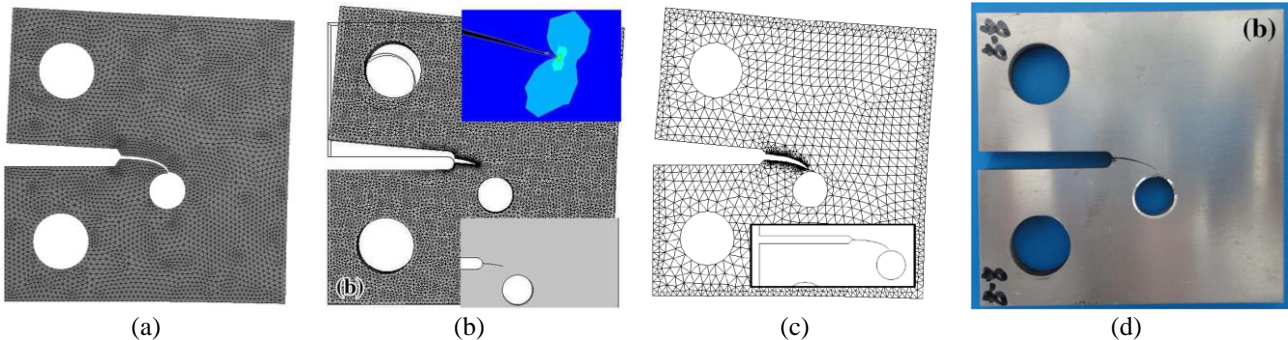


Figure 7: G3 deformed shape, equivalent von mises, and crack path: a) Ansys, b) Present developed program, c) Franc3D [24], d) Experimental work [24].

In order to see the effect of the hole size, the size of d_1 was reduced to 2 mm in diameter for the G5 specimen. The h_1 and v_1 is identical to that of G2. The fatigue crack growth can be seen to be almost straight but when it comes in the range of the 2 mm hole, it deviates slightly and then returns to its original path as shown in Figure 11. The programs show a good agreement with the experimental work of [24]. Number of cycles computed in Ansys was 52116 with a crack length of 8.8 mm while in Franc3D it was 6018 cycles and 9.1 mm [24].

For specimen G5, in Present developed program the number of nodes and elements are 5448 and 2566, respectively. While in Ansys it is 175255 nodes and 120570 elements. The range of K_I in Present developed program is between 577 and 880 $\text{MPa}\cdot\text{mm}^{0.5}$, Ansys has a range from 544 to 857 $\text{MPa}\cdot\text{mm}^{0.5}$ as shown in Figure 12. The difference in the SIF values is 3.2 %.

For the specimen G6, the size of d_1 is 4 mm and has h_1 of 6.5 mm. From Figure 13 it can be noticed that the fatigue crack path as in G5 is influenced slightly by the hole but continues back in an approximately straight line. All three programs predicted the right crack path as shown in comparison with the experimental work. The graphical results for Franc3D displayed in Figure 13(c) show that the number cycles come to 5519 cycles and the stable crack length of the path is 9.8 mm while for Ansys it was 44968 cycles and a crack length of 9.0427 mm. Figure 14 shows the predicted stress intensity factor modes one and two obtained in both Present developed program and Ansys. The K_I range in Present developed program is from 575 to 760 $\text{MPa}\cdot\text{mm}^{0.5}$ while for Ansys it is from 467 to 821 $\text{MPa}\cdot\text{mm}^{0.5}$. Furthermore, the difference of these values obtained in both programs is 10.4%. This difference is large and is due to

mesh density. For Ansys the geometry has a generated mesh of 170240 nodes and 116364 elements. While in Present developed program, 5137 nodes and 2411 elements.

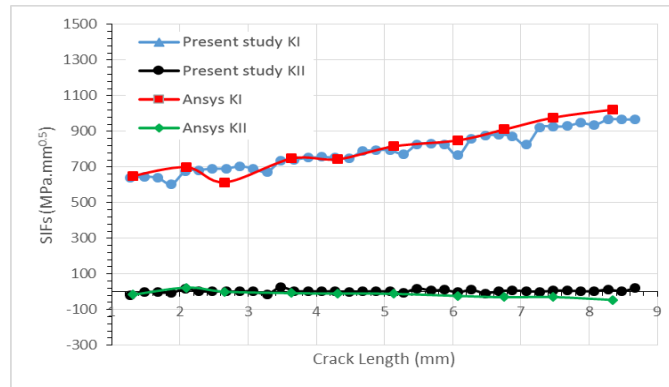


Figure 8: Specimen G3 simulation, SIFs vs crack length.

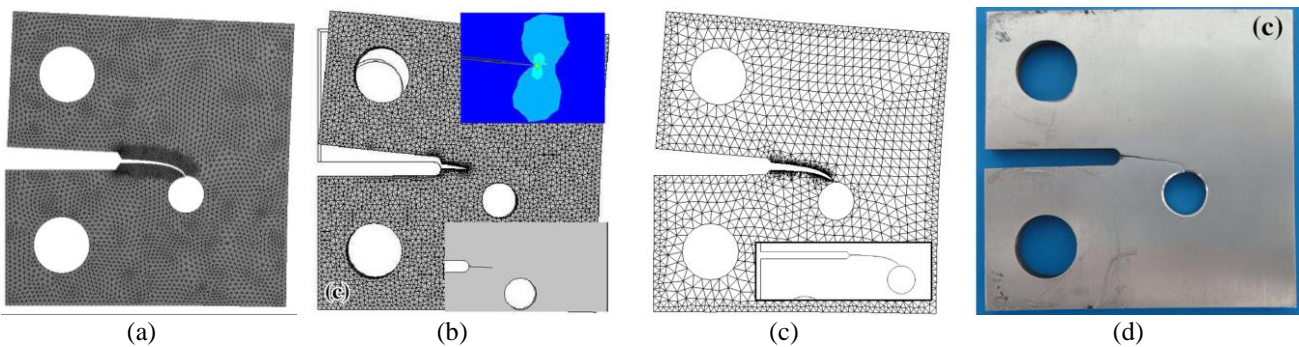


Figure 9: G4 deformed shape, equivalent von mises and crack path: a) Ansys, b) Present developed program, c) Franc3D [24], d) Experimental work [24].

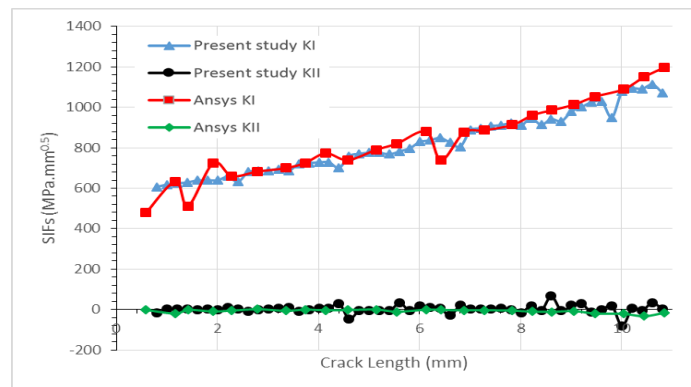


Figure 10: Mode I and II relation with crack length in specimen G4.

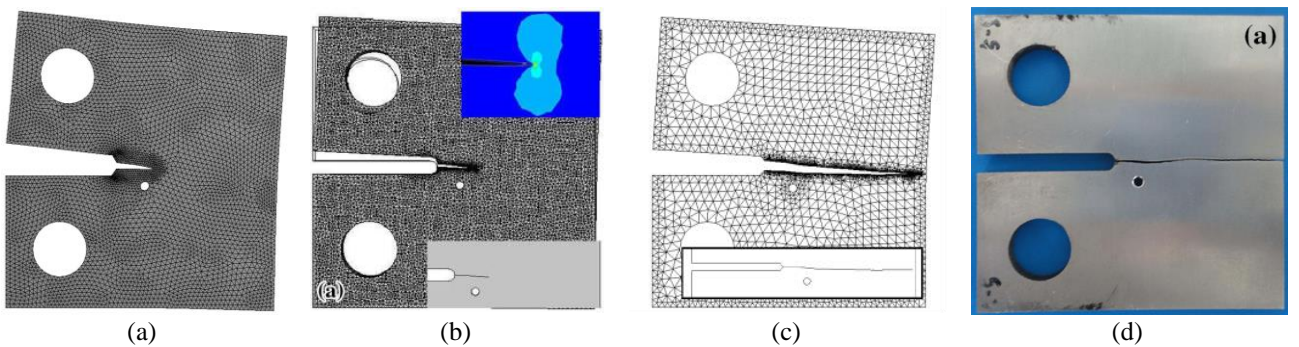


Figure 11: G5 deformed shape, equivalent von mises, and crack path: a) Ansys, b) Present developed program, c) Franc3D [24], d) Experimental work respectively [24]

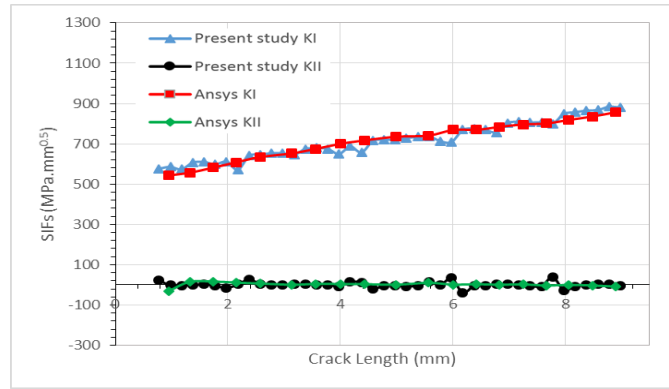


Figure 12: SIFs relation with the crack length in specimen G5.

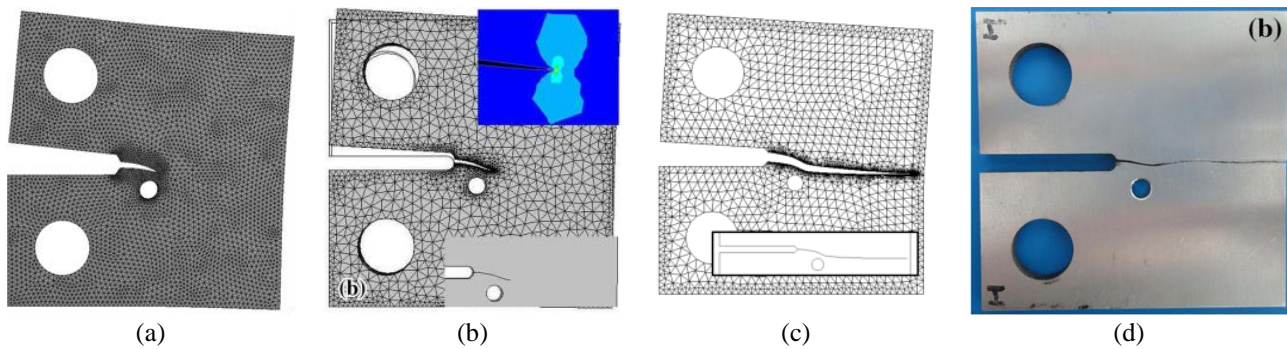


Figure 13: G6 deformed mesh, equivalent von mises, and crack path: a) Ansys, b) Present developed program, c) Franc3D [24], d) Experimental work [24]

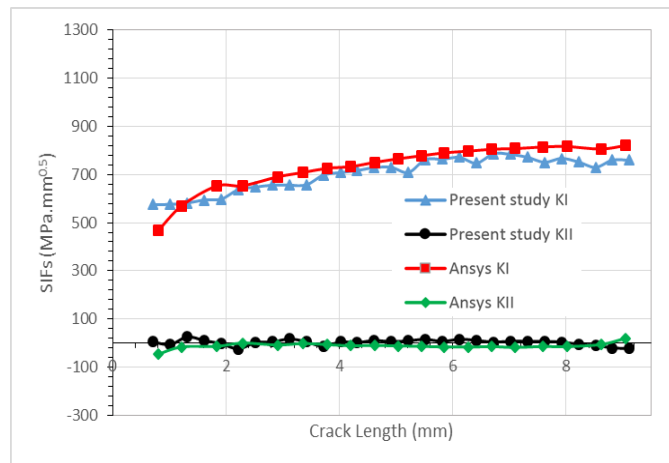


Figure 14: The computed K_I and K_{II} relationship with the crack length in specimen G6

To get a better picture on the effects of the hole on the crack path, two holes are introduced in specimen G7. The holes are of 5 mm in diameter and at equal spacings from each other and the crack front. In the following condition, the number of cycles obtained in Ansys is 57762 and a crack length of 19.638 mm while for Franc3D it was 7211 cycles and a crack growth of 10 mm. Figure 15 shows that the crack path for in this specimen is almost straight just like in G1, but what changed is the number of cycles and fatigue life of the specimen which increased drastically.

G7, a geometry with two 5 mm diameter holes at equal distances from other and along the crack path has a K_I range from 640 to 898 $\text{MPa}\cdot\text{mm}^{0.5}$ computed in Present developed program while in Ansys the mode 1 ranged from 526 to 898 $\text{MPa}\cdot\text{mm}^{0.5}$ as shown in Figure 16. The crack length obtained in Present developed program was 10.37 mm and in Ansys 10.7 mm. The number of nodes and elements by which the mesh was generated in Present developed program are 4970 and 2333, respectively. For Ansys it is 88182 and 59154, respectively.

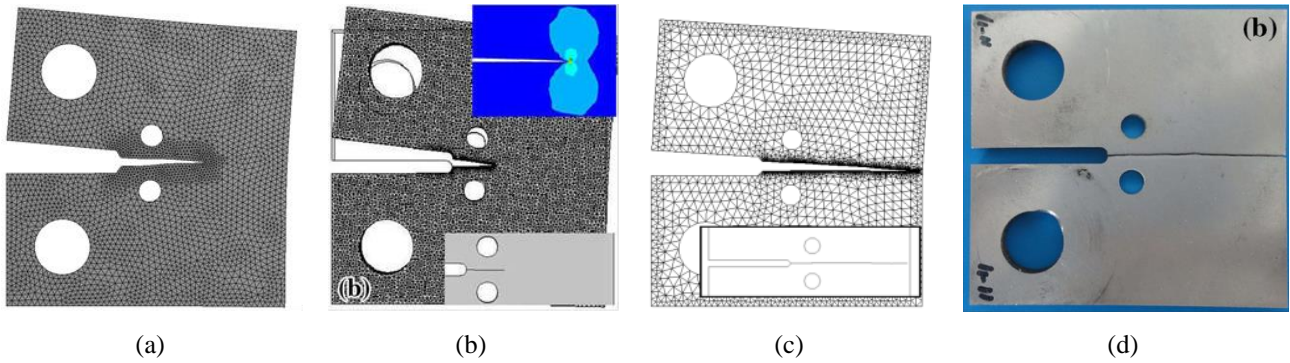


Figure 15: G7 deformed shape, equivalent von mises, crack path: a) Ansys, b) Present developed program, c) Franc3D [24], d) Experimental work [24]

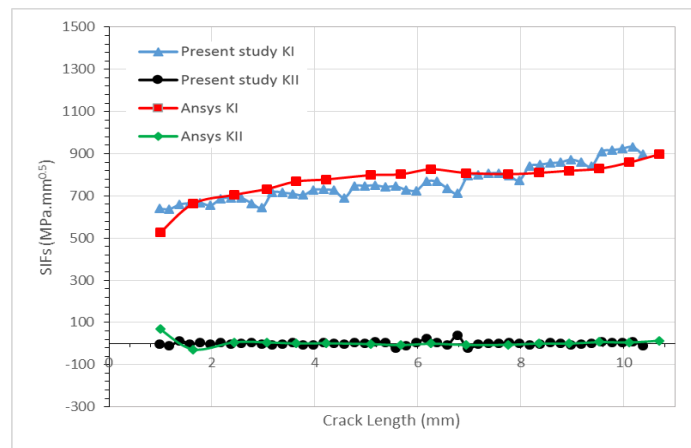


Figure 16: G7, K_I and K_{II} with crack length.

Conclusion

It can be concluded that there is almost slightly difference in performing a fracture mechanics simulation for crack growth in both two- or three- dimensions. Sometimes it wouldn't be ideal to perform a simulation on a complex geometry in 2D and hence the need for 3D. Two dimensional analysis provides a better computational time than 3D as well as giving the ability to generate a more finer mesh on the geometry due to available computational power. The predicted values of stress intensity factors in both 2D and 3D are almost the same with a slight percentage of error which can be eliminated by using higher mesh density in the two-dimensional program. The holes size and position are affected the crack trajectory as well as the values of stress intensity factors.

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